

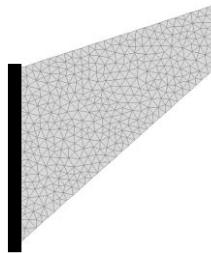
# TD2 EX1 : Compute forces on $S_u$

1- Matlab code lines to compute forces  $\underline{R} = R_1 \underline{e}_1 + R_2 \underline{e}_2$  on  $S_u$

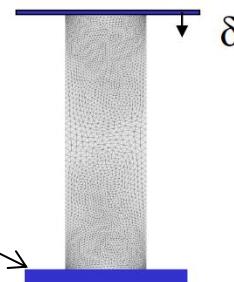
with, by definition  $\underline{R} = \int_{S_u} \underline{\sigma} \cdot \underline{n} dS$  and  $R_1 = \left( \int_{S_u} \underline{\sigma} \cdot \underline{n} dS \right) \cdot \underline{e}_1$   $R_2 = \left( \int_{S_u} \underline{\sigma} \cdot \underline{n} dS \right) \cdot \underline{e}_2$

2- Applications: wing & compress

**Wing case:**  
on  $S_u$  ( $x=0$ )  
 $(R_1, R_2) = ?$



**Compress case:**  
on  $S_u$  ( $y=0$ )  
 $(R_1, R_2) = ?$



# Hints :TD2 EX1 : Compute forces on $S_u$

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## Indication:

Use PPV (book: eq. 1.19) with a well-chosen test field  $\underline{w}_h \in C$

$$\int_{\Omega} \underline{\underline{\varepsilon}}[\underline{u}_h] : \mathcal{A} : \underline{\underline{\varepsilon}}[\underline{w}_h] dV = \int_{\Omega} \rho \underline{f} \cdot \underline{w}_h dV + \int_{S_T} \underline{T}^D \cdot \underline{w}_h dS + \int_{S_u} [\underline{\underline{\sigma}} \cdot \underline{n}] \cdot \underline{w}_h dS$$

To find nodes on  $S_u ==>$  use matlab function « find »

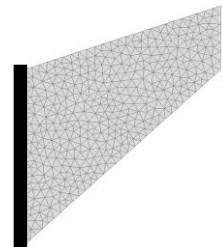
`S_nodes = find(coor(:,1)<(1.d-4));` ==>  $S_{nodes}$  returns the indices of « coor(:,1) » that satisfy the condition

2- Applications: wing & compress

Wing case:

on  $S_u$  ( $x=0$ )

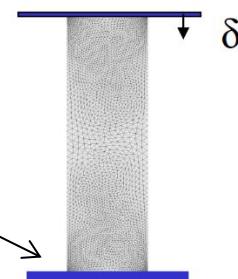
$(R_1, R_2) = ?$



Compress case:

on  $S_u$  ( $y=0$ )

$(R_1, R_2) = ?$



# TD2 EX2 Axisymmetric element (T6axi)

Axisymmetric problems :

- + 3D geometry is completely represented by a “2D” mesh
- + Nothing depends on  $\theta$

==> axisymmetric formulation

1- Writes PPV (TTV) in function of r and z

$$\underline{u}(r, z) = u_r(r, z)\underline{e}_r + u_z(r, z)\underline{e}_z$$

$$\underline{\varepsilon}(\underline{u}) = u_{r,r}\underline{e}_r \otimes \underline{e}_r + u_{z,z}\underline{e}_z \otimes \underline{e}_z + \frac{u_r}{r}\underline{e}_\theta \otimes \underline{e}_\theta + \frac{1}{2}(u_{r,z} + u_{z,r})(\underline{e}_r \otimes \underline{e}_z + \underline{e}_z \otimes \underline{e}_r)$$

2- Definition of  $\{\varepsilon\}$  et  $\{\sigma\}$  ==> [A] :  $\{\sigma\} = [A]\{\varepsilon\}$

3- Definition of [A] ==> utility/read\_input.m (line 263)

4- Create in T6axi directory 3 functions:

- stiff\_linel\_T6axi.m (local stiffness matrix)
- nf\_tractions\_T6axi.m (local generalized force vector)
- stressG\_linel\_T6axi.m (Cauchy stress component at Gauss points)

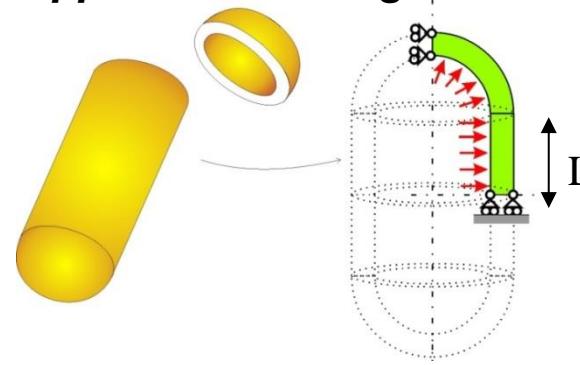
5- Analysis / analytical comparaison

**Analytical solution:**

$$\text{Sphere under pressure: } u_r = A \frac{(1-2\nu)}{E} r + B \frac{1+\nu}{E} \frac{1}{r^2}, \quad A = p \frac{R_I^3}{R_E^3 - R_I^3}, \quad B = p \frac{R_I^3 R_E^3}{2(R_E^3 - R_I^3)}$$

$$\text{Cylinder under pressure: } u_r = A \frac{(1-2\nu)}{E} r + B \frac{(1+\nu)}{E} \frac{1}{r}, \quad A = p \frac{R_I^2}{R_E^2 - R_I^2}, \quad B = p \frac{R_I^2 R_E^2}{R_E^2 - R_I^2}$$

**Application: diving bottle**



**Geometry:**

$$R_I = 10, R_E = 11, L = 20$$

**Loading:**

internal pressure  $p=1$

**Input File :**

input/reservoir.geo,  
input/reservoir.inp

$$\text{A.N. : } U_{\text{sphère}} = 38.2$$

$$U_{\text{cylindre}} = 94$$

**Send by email:** ([clement.olivier@safran.fr](mailto:clement.olivier@safran.fr))

- Your names
- 4 matlab programs: **stiff\_linel\_T6axi.m, nf\_tractions\_T6axi.m, stressG\_linel\_T6axi.m** 1-2 pages (max) of comments and analysis